

## Viscous Flow lecture 3

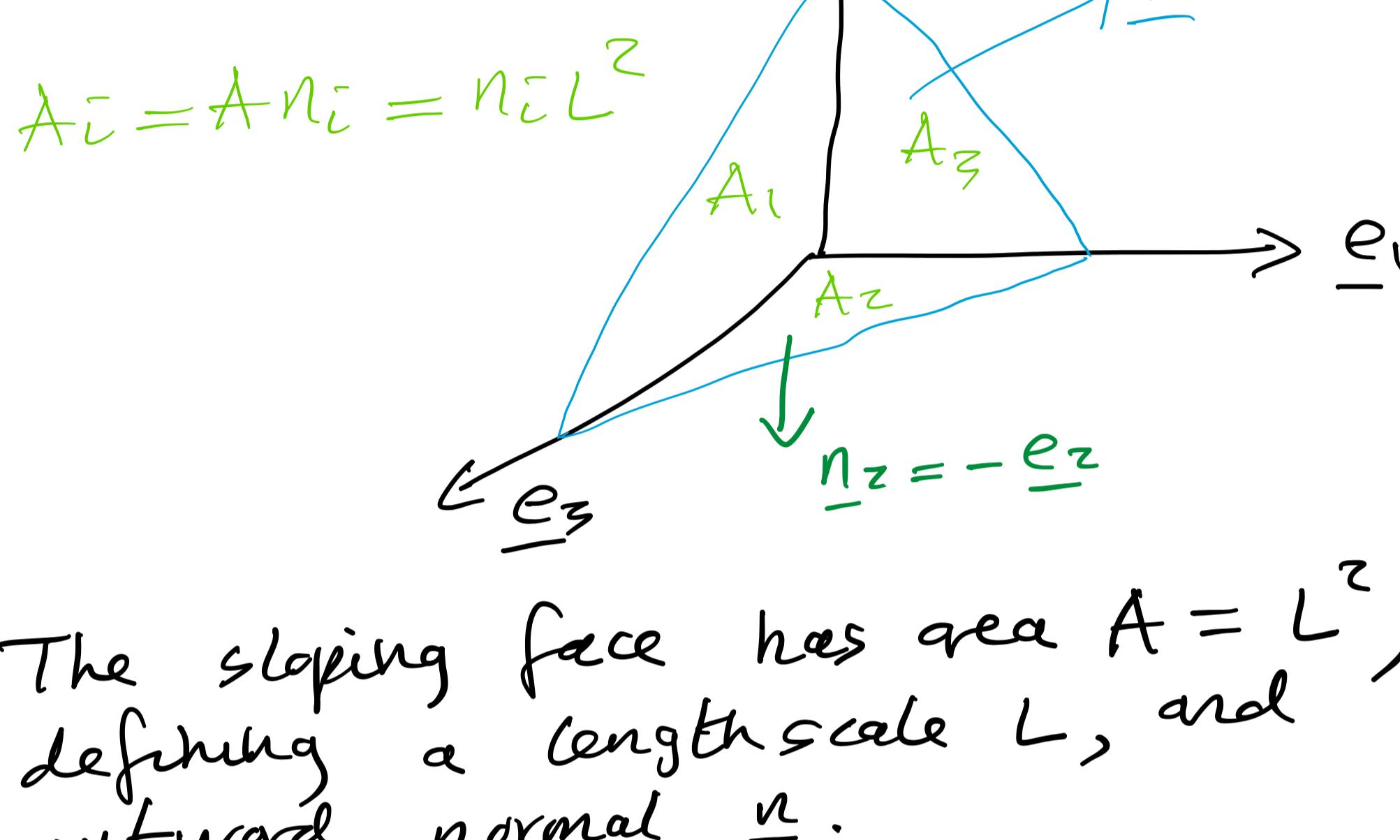
Last time: Newton's 3rd law

$$\underline{\underline{\tau}}(\underline{n}) = -\underline{\underline{\tau}}(-\underline{n})$$

Cauchy's Stress Theorem

$$\underline{\underline{\tau}}(\underline{n}) = \underline{e}^i \sigma_{ij} n_j$$

Proof: Consider a material volume  $V(t)$  that is instantaneously a tetrahedron as drawn:



The sloping face has area  $A = L^2$ , defining a length scale  $L$ , and outward normal  $\underline{n}$ .

The three faces  $A_i$  in the planes  $x_i = 0$  have areas  $A_i = A n_i = n_i L^2$ .

$$NII \Rightarrow \iiint_{V(t)} \rho \frac{D\underline{u}}{Dt} - \rho \underline{F} dV = \iint_{\partial V(t)} \underline{\underline{\tau}}(\underline{n}) dS$$

LHS  $\rightarrow O(L^3)$  as  $L \rightarrow 0$ , assuming the integrand is continuous, hence bounded, in  $V(t)$ . implied sum over j

$$RHS = \underline{\underline{\tau}}(\underline{n}) L^2 + \underline{\underline{\tau}}(-\underline{e}_j) \underline{n}_j L^2 + O(L^3)$$

where  $\underline{\underline{\tau}}$  is evaluated at  $\underline{x} = 0$  to sufficient accuracy.

Taking  $L \rightarrow 0$  establishes that

$$\underline{\underline{\tau}}(\underline{n}) + \underline{\underline{\tau}}(-\underline{e}_j) \underline{n}_j = 0.$$

$$NIII \Rightarrow \underline{\underline{\tau}}(-\underline{e}_j) = -\underline{\underline{\tau}}(\underline{e}_j) = -\underline{e}^i \sigma_{ij}$$

$$\therefore \underline{\underline{\tau}}(\underline{n}) - \underline{e}^i \sigma_{ij} \underline{n}_j = 0$$

This is Cauchy's stress theorem.

Again, the net force on an arbitrarily small volume must vanish to avoid an infinite acceleration.

Given 9 quantities  $\sigma_{ij}$  we can compute

$$\underline{\tau}(\underline{n}) = \underline{e}_i \sigma_{ij} n_j$$

for any direction  $\underline{n}$  of the normal.

Now we can convert that surface stress integral into a volume integral.

$$\iint_{\partial V(t)} \underline{\tau}(\underline{n}) dS = \iint_{V(t)} \underline{e}_i \sigma_{ij} n_j dS$$

$$= \underline{e}_i \iiint_{V(t)} \frac{\partial}{\partial x_j} \sigma_{ij} dV$$

by the divergence theorem.

is why  
some  
people  
define  
this as

$$\frac{\partial}{\partial x_j} \sigma_{ji}$$

NII for  $V(t)$  becomes

$$\iint_{V(t)} \rho \frac{D\underline{u}}{DT} - \underline{e}_i \frac{\partial \sigma_{ij}}{\partial x_j} - \rho \underline{F} dV = 0$$

True for all material volumes  $V(t)$  so

$$\rho \frac{D\underline{u}}{DT} = \underline{e}_i \frac{\partial \sigma_{ij}}{\partial x_j} + \rho \underline{F}$$

For inviscid fluids,  $\sigma_{ij} = -P \delta_{ij}$

so we recover the Euler equation

$$\rho \frac{D\underline{u}}{DT} = -\nabla P + \rho \underline{F}$$

$\int \int \underline{t}(\underline{n}) dS$  looks like it should  
 $\frac{\partial V(t)}{\partial}$  be proportional to  
 the area  $| \partial V(t) | = O(\epsilon)$

We've shown that

$$\int \int \underline{t}(\underline{n}) dS = \int \int \int \underline{e}_i \frac{\partial \sigma_{ij}}{\partial x_j} dV$$

$$\frac{\partial V(t)}{\partial}$$

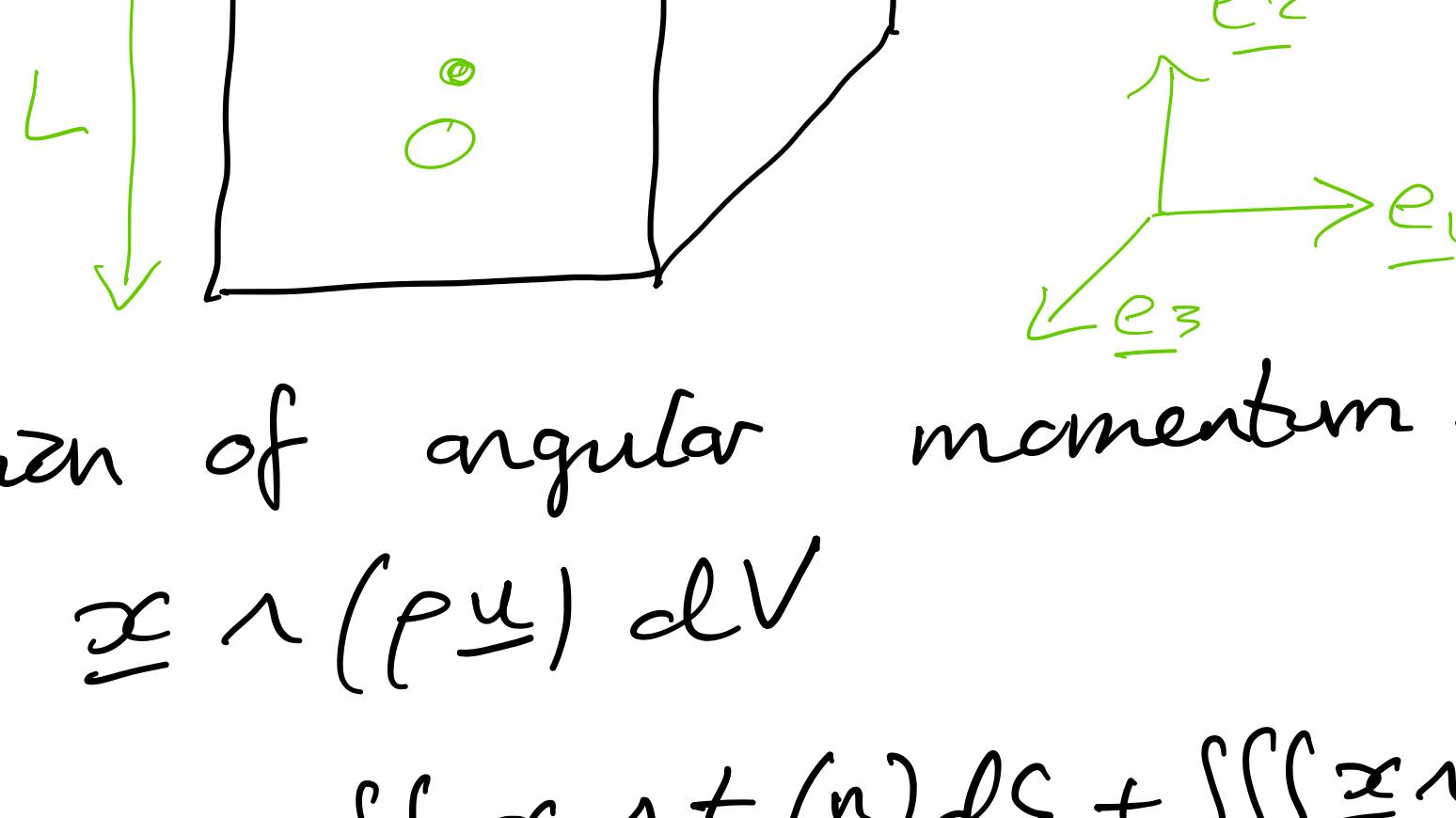
is in fact proportional to the volume  $L^3$ .

If  $\sigma_{ij}$  is constant, the integral vanishes — the flux in exactly balances the flux out.

## Properties of $\sigma_{ij}$ in general

① It is symmetric,  $\sigma_{ij} = \sigma_{ji}$

Proof: Consider a small material volume  $V(t)$  instantaneously forming a cube centred at  $O$  with faces  $x_j = \pm \frac{L}{2}$ .



conservation of angular momentum:

$$\frac{d}{dt} \iint \iint \frac{\rho u}{V(t)} dS = \iint \frac{\rho u}{\partial V(t)} dS + \iint \frac{\rho E}{V(t)} dV$$

Applying RTT with  $f = e_i \cdot (\rho u)$

$$\begin{aligned} \frac{d}{dt} \iint \iint \frac{\rho u}{V(t)} dS &= \iint \frac{\rho \frac{Du}{DE} - \rho E}{\partial V(t)} dS \\ &= O(L^4) \end{aligned}$$

$$RHS = \frac{L}{2} e_j \wedge L^2 t(e_j)$$

$$+ (-\frac{L}{2} e_j) \wedge L^2 t(-e_j)$$

$$+ O(L^4)$$

Again evaluating  $t$  at the origin  $O$  to sufficient accuracy.

$$NIII \Rightarrow t(-e_j) = -t(e_j)$$

$$so RHS = L^3 e_j \wedge t(e_j) + O(L^4)$$

$$LHS = O(L^4).$$

This has to be true for arbitrarily small  $L$ , so  $e_j \wedge t(e_j) = 0$

We know  $t(e_j) = e_i \sigma_{ij}$  by Cauchy's Stress Theorem so

$$e_j \wedge e_i \sigma_{ij} = 0.$$

$$e_1 (\sigma_{32} - \sigma_{23}) + e_2 (\sigma_{13} - \sigma_{31})$$

$$+ e_3 (\sigma_{21} - \sigma_{12}) = 0$$

$\therefore \sigma_{ij} = \sigma_{ji} \Rightarrow$  symmetric.

Alternatively:  $e_j \wedge e_i$  is antisymmetric in swapping  $i \leftrightarrow j$

$$\therefore e_j \wedge e_i = -e_i \wedge e_j$$

$\therefore \sigma_{ij}$  must be symmetric to make

$$\underbrace{e_j \wedge e_i}_{\text{antisymmetric}} \underbrace{\sigma_{ij}}_{\text{symmetric}} = 0$$

antisymmetric symmetric

$\sigma_{ij}$  are the components of a tensor

Proof:  $\sigma_{ij}$  relates two vectors  $\underline{t} \in \mathbb{R}^n$   
via  $\underline{t}(n) = \sum_i \sigma_{ij} n_j$

In components:  $t_i = \sigma_{ij} n_j$

Consider a rotation from axes

$Ox_1 x_2 x_3$  with basis  $e_1, e_2, e_3$

to  $Ox'_1 x'_2 x'_3$  with basis  $e'_1, e'_2, e'_3$ .

For any vector,  $\underline{x} = x_j e_j = x'_i e'_i$   
 $x'_i = e'_i \cdot \underline{x} = e'_i \cdot e_j x_j = L_{ij} x_j$

$L_{ij} = e'_i \cdot e_j$  are the components  
of an orthogonal matrix  $L$ , so  $LL^T = I$ .

$$\therefore x_j = (L^T)_{ji} x'_i = L_{ij} x'_i$$

Applying these transformations to  
Cauchy's stress theorem:

$$t'_i = L_{ij} t_j = L_{ij} \sigma_{jk} n_k \quad ①$$

$$\text{and } t'_i = \sigma'_{ij} n'_j = \sigma'_{ij} L_{jk} n_k \quad ②$$

Comparing ① & ② for all  $n_k$

$$\Rightarrow L_{ij} \sigma_{jk} = \sigma'_{ij} L_{jk}$$

In matrix notation

$$L \sigma = \sigma' L$$

Multiply on the right by  $L^T$  &  
get

$$L \sigma L^T = \sigma'$$

so  $\sigma$  transforms like a (rank 2)  
tensor, compare with

$$x'_i = L_{ij} x_j$$

$$\underline{x}' = L \underline{x}$$

Think about how the tensors  
 $t_i t_j$  or  $n_i n_j$  transform  
given we know how the vectors  
 $t_i$  and  $n_i$  transform.