

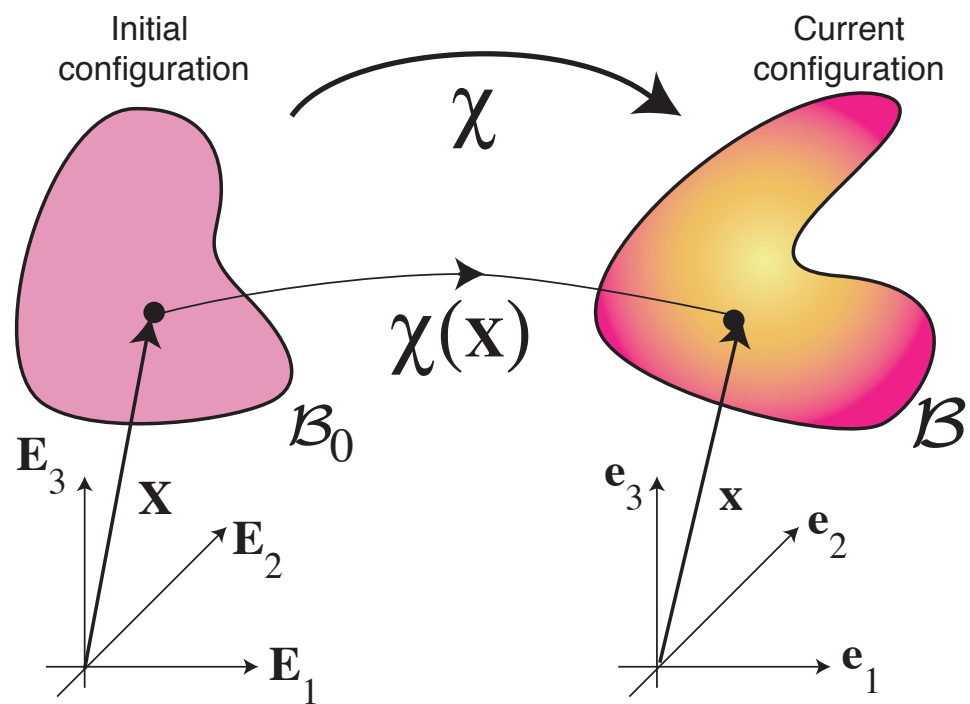
SOLID MECHANICS

Lecture 8: Chapter 3: Dynamics

Section 3.2: Balance laws-transport formula

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Tensor calculus ϕ , \mathbf{v} , \mathbf{T} are, scalar, vector and 2^{nd} -order tensor fields

$$\mathbf{F} = \text{Grad } \mathbf{x} = \frac{\partial x_i}{\partial X_j} \mathbf{e}_i \otimes \mathbf{E}_j \quad \text{Deformation Gradient} \quad (T1)$$

$$J = \det \mathbf{F} \quad \text{Determinant of } \mathbf{F} \quad (T2)$$

$$\text{grad } \mathbf{v} = \frac{\partial \mathbf{v}}{\partial x_i} \otimes \mathbf{e}_i \quad \text{Definition of the gradient of a vector} \quad (T3)$$

$$\text{grad } \mathbf{T} = \frac{\partial \mathbf{T}}{\partial x_i} \otimes \mathbf{e}_i \quad \text{Definition of the gradient of a tensor} \quad (T4)$$

$$\text{div } \mathbf{T} = \frac{\partial T_{ij}}{\partial x_i} \mathbf{e}_j \quad \text{Definition of the divergence of a tensor} \quad (T5)$$

$$\text{Grad } \phi = \mathbf{F}^T \text{grad } \phi \quad \text{Gradients of a scalar} \quad (T6)$$

$$\text{Grad } \mathbf{v} = (\text{grad } \mathbf{v}) \mathbf{F} \quad \text{Gradients of a vector} \quad (T7)$$

$$\text{Div } \mathbf{v} = J \text{div} (J^{-1} \mathbf{F} \mathbf{v}) \quad \text{Divergences of a vector} \quad (T8)$$

$$\text{Div } \mathbf{T} = J \text{div} (J^{-1} \mathbf{F} \mathbf{T}) \quad \text{Divergences of a tensor} \quad (T9)$$

$$\text{div} (J^{-1} \mathbf{F}) = 0 \quad \text{An important identity} \quad (T10)$$

$$\frac{\partial}{\partial \lambda} (\det \mathbf{T}) = (\det \mathbf{T}) \text{tr} \left(\mathbf{T}^{-1} \frac{\partial \mathbf{T}}{\partial \lambda} \right) \quad \text{A useful identity. } \lambda \text{ is a scalar} \quad (T11)$$

Kinematics

$$\mathbf{F} = \text{Grad } \mathbf{x}(\mathbf{X}, t) \quad \text{The deformation gradient} \quad (K1)$$

$$J = \det \mathbf{F} \quad \text{Determinant of } \mathbf{F} \quad (K2)$$

$$d\mathbf{x} = \mathbf{F}d\mathbf{X} \quad \text{Transformation of line element} \quad (K3)$$

$$d\mathbf{a} = J\mathbf{F}^{-\top}d\mathbf{A} \quad \text{Transformation of area element} \quad (K4)$$

$$dv = JdV \quad \text{Transformation of volume element} \quad (K5)$$

$$\mathbf{C} = \mathbf{F}^{\top}\mathbf{F} \quad \text{Right Cauchy-Green tensor} \quad (K6)$$

$$\mathbf{B} = \mathbf{F}\mathbf{F}^{\top} \quad \text{Left Cauchy-Green tensor} \quad (K7)$$

$$\mathbf{E} = \frac{1}{2} (\mathbf{F}^{\top}\mathbf{F} - \mathbf{1}) \quad \text{Euler strain tensor} \quad (K8)$$

$$\mathbf{L} = \text{grad } \mathbf{v} \quad \text{Velocity gradient} \quad (K9)$$

$$\dot{\mathbf{F}} = \mathbf{L}\mathbf{F} \quad \text{Evolution of the deformation gradient}(\mathbf{v} : \text{velocity}) \quad (K10)$$

$$\dot{J} = J\text{div } \mathbf{v} \quad \text{Evolution of the volume element} \quad (K11)$$

$$\mathbf{D} = \frac{1}{2} (\mathbf{L} + \mathbf{L}^{\top}) \quad \text{Eulerian strain rate tensor} \quad (K12)$$

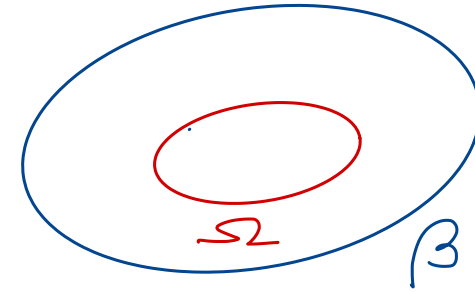
$$\mathbf{W} = \frac{1}{2} (\mathbf{L} - \mathbf{L}^{\top}) \quad \text{Rate of rotation tensor} \quad (K13)$$

3 Conservation Laws, Stress, and Dynamics

1.1 Balance of linear momentum

1.1.1 Body and surface forces

Consider $\Omega \subseteq \mathcal{B}$.



Define $\mathbf{F}(\Omega)$: *total force* acting on Ω , and $\mathbf{G}(\Omega, \mathbf{0})$: *total torque* acting on Ω with respect to the origin $\mathbf{0}$

Total body forces \mathbf{F} due to body forces (external) and contact forces (internal)

Body forces and torques. Define $\mathbf{b}(\mathbf{x}, t)$ the *body force per unit mass* and $\mathbf{c}(\mathbf{x}, t)$ the *body torque per unit mass* with respect to $\mathbf{0}$.

$$F_{\text{body}} = \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{b}(\mathbf{x}, t) dv \quad (1)$$

$$G_{\text{body}}(\Omega, \mathbf{0}) = \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{c}(\mathbf{x}, t) dv + \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{x} \times \mathbf{b}(\mathbf{x}, t) dv \quad (2)$$

Assumption: For the rest of the course: non-polar medium $\mathbf{c} = \mathbf{0}$.

$\rho(\vec{x}, t)$: density (mass/current volume)

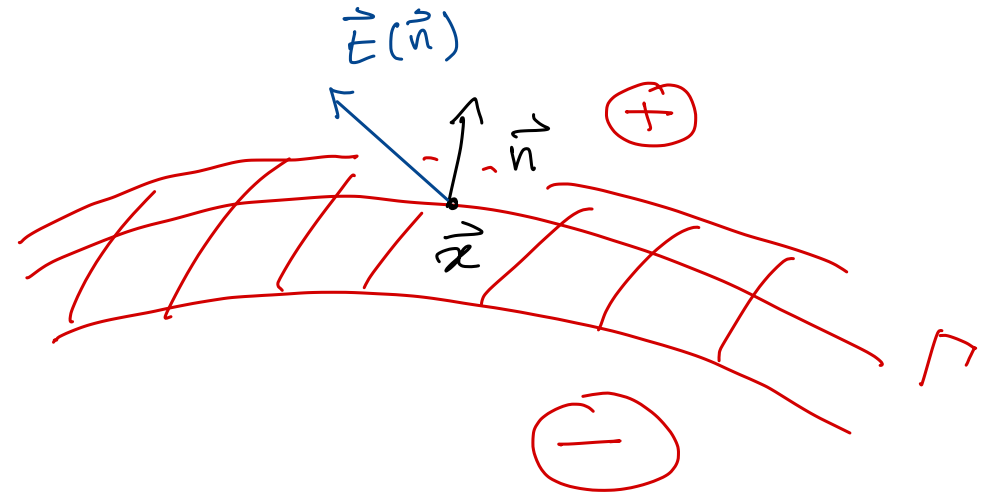
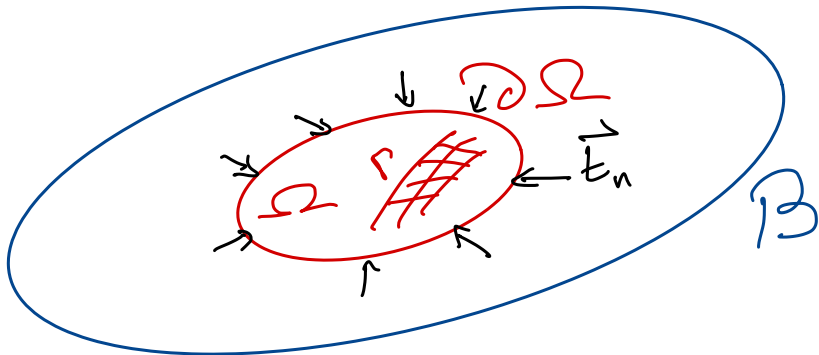
Contact forces and torques.

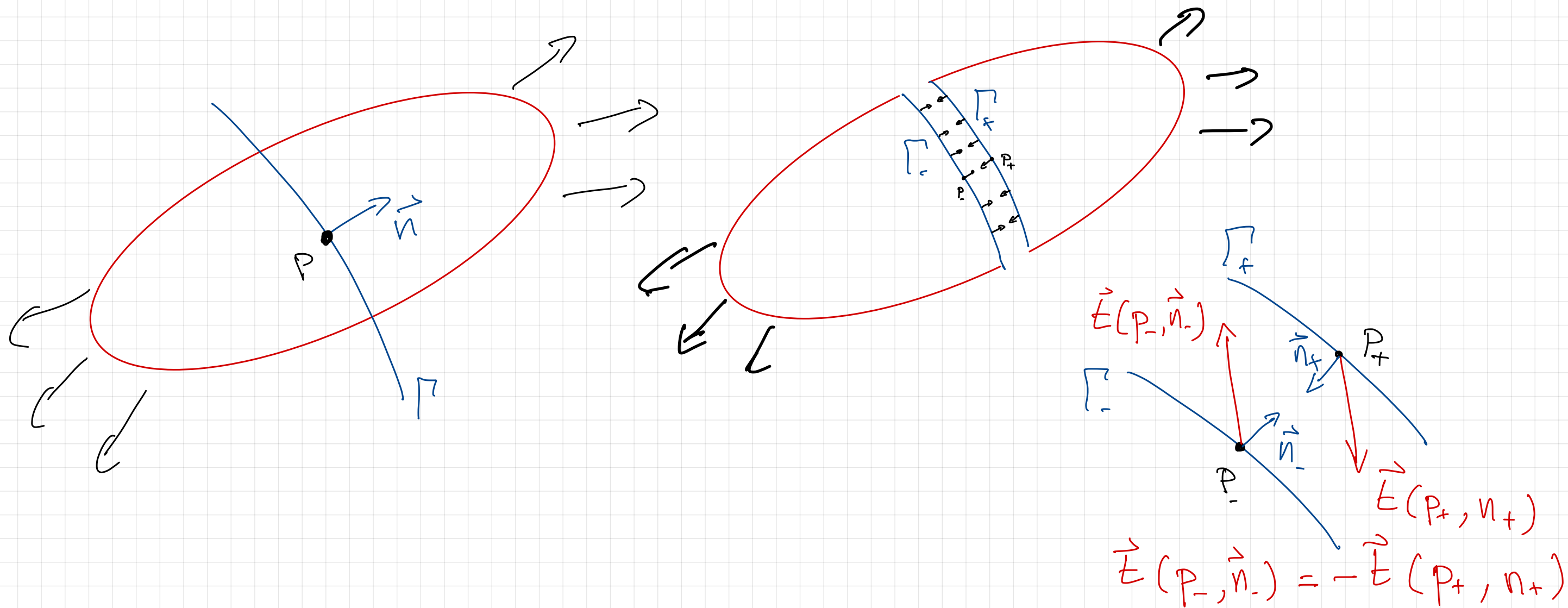
$$F_{\text{contact}} = \int_{\partial\Omega} \mathbf{t}_n \, da \quad (3)$$

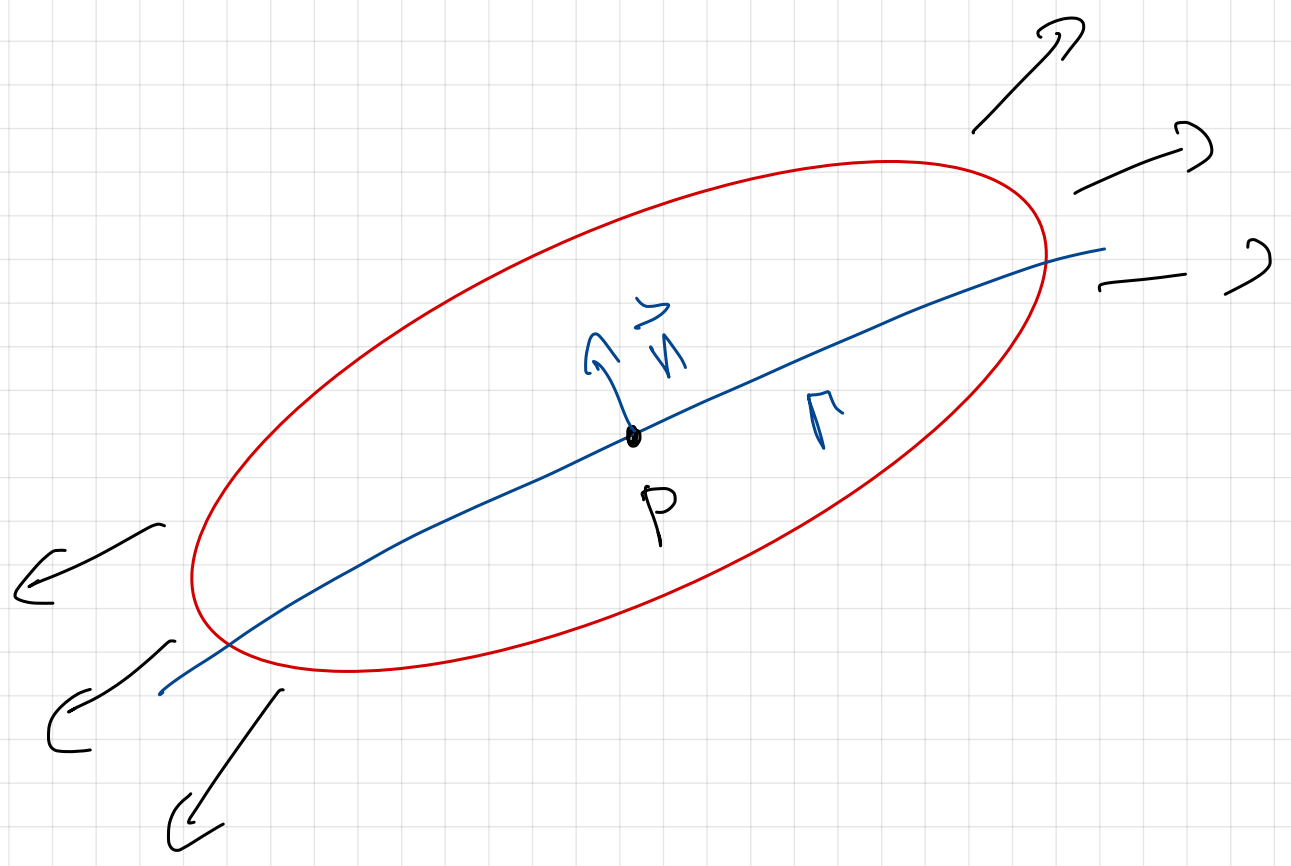
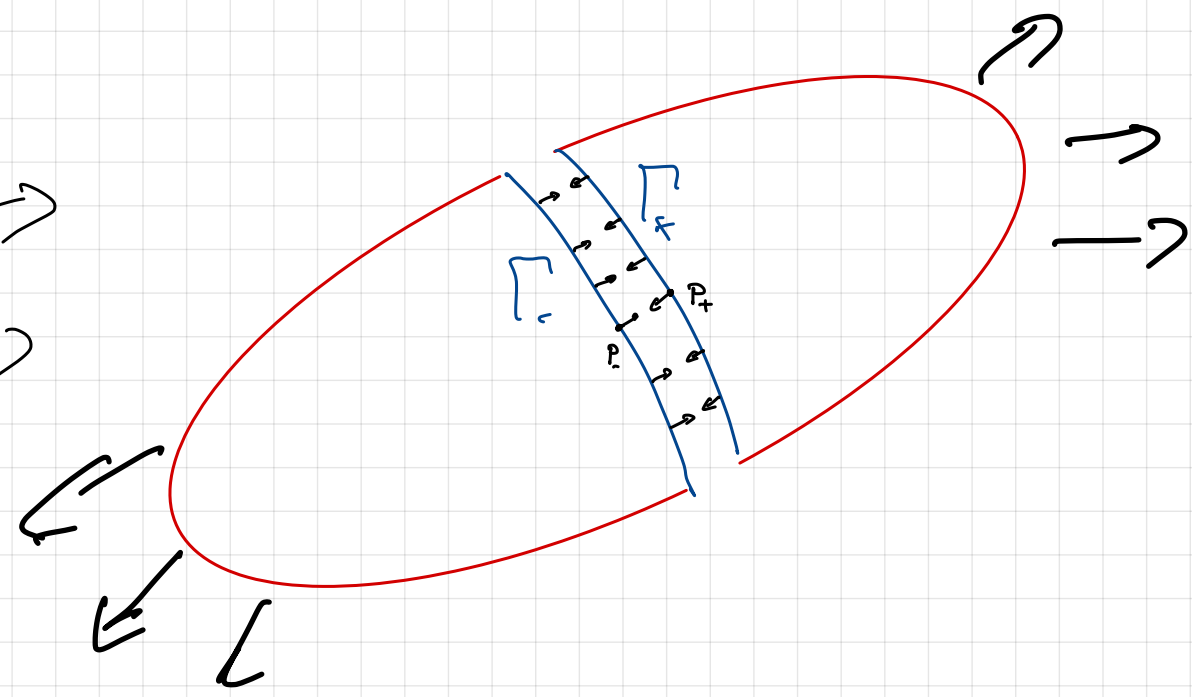
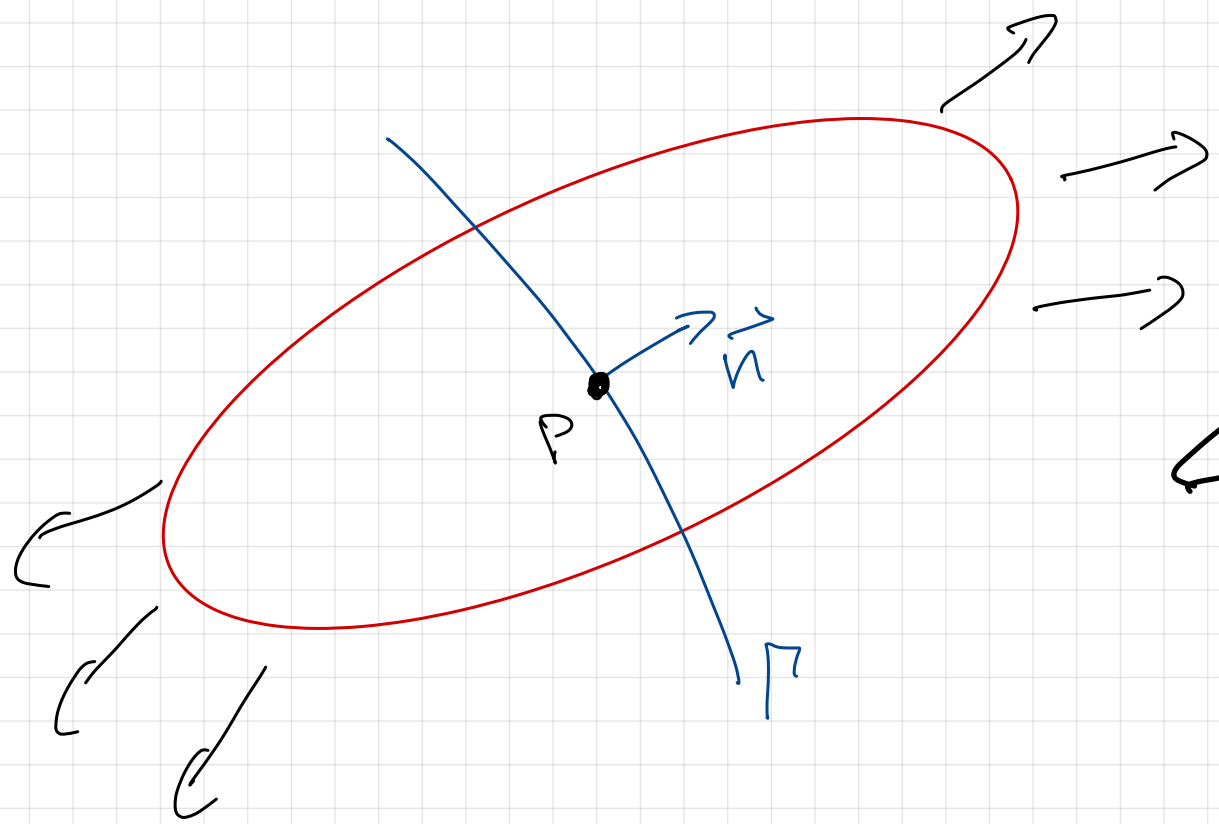
$$G_{\text{contact}}(\Omega, \mathbf{0}) = \int_{\partial\Omega} \mathbf{x} \times \mathbf{t}_n \, da. \quad (4)$$

where \mathbf{t}_n is the *traction* (force per unit area).

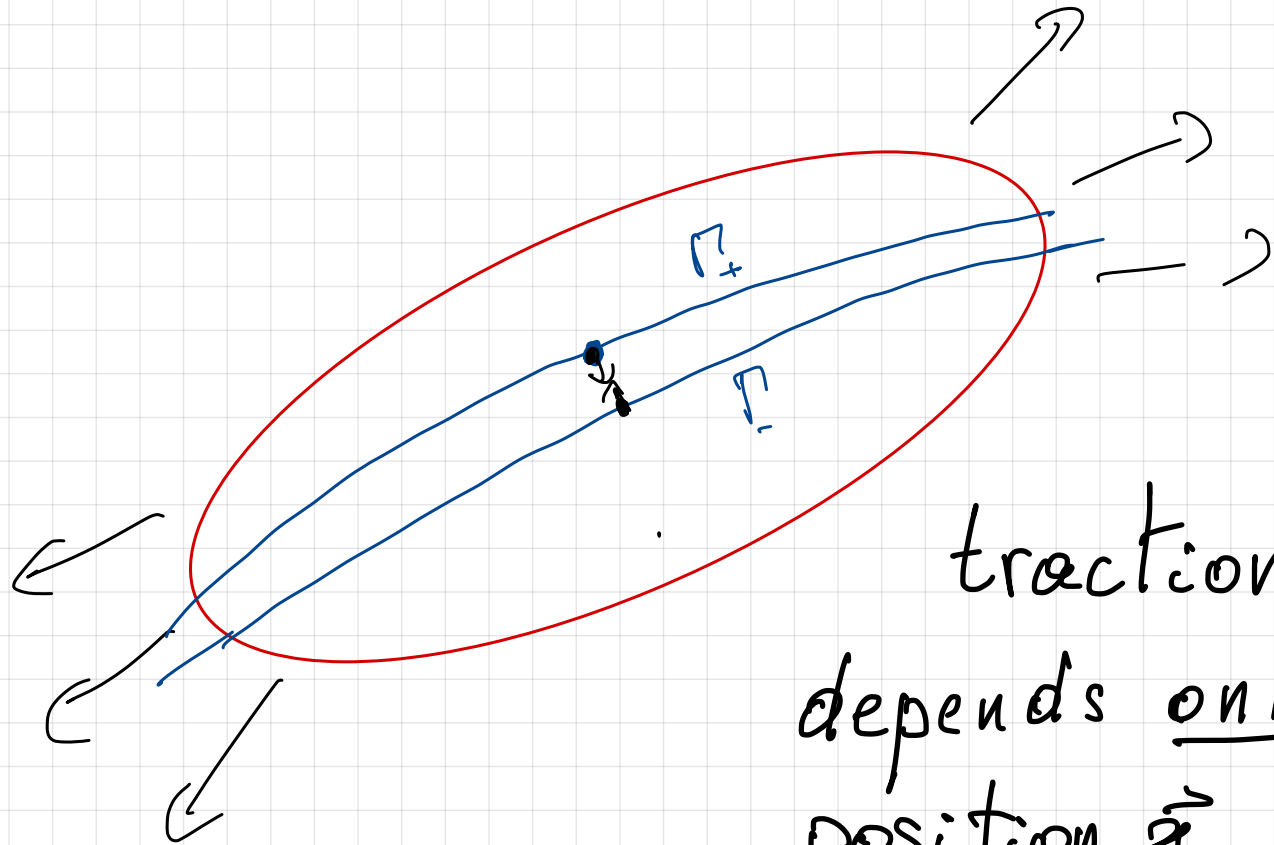
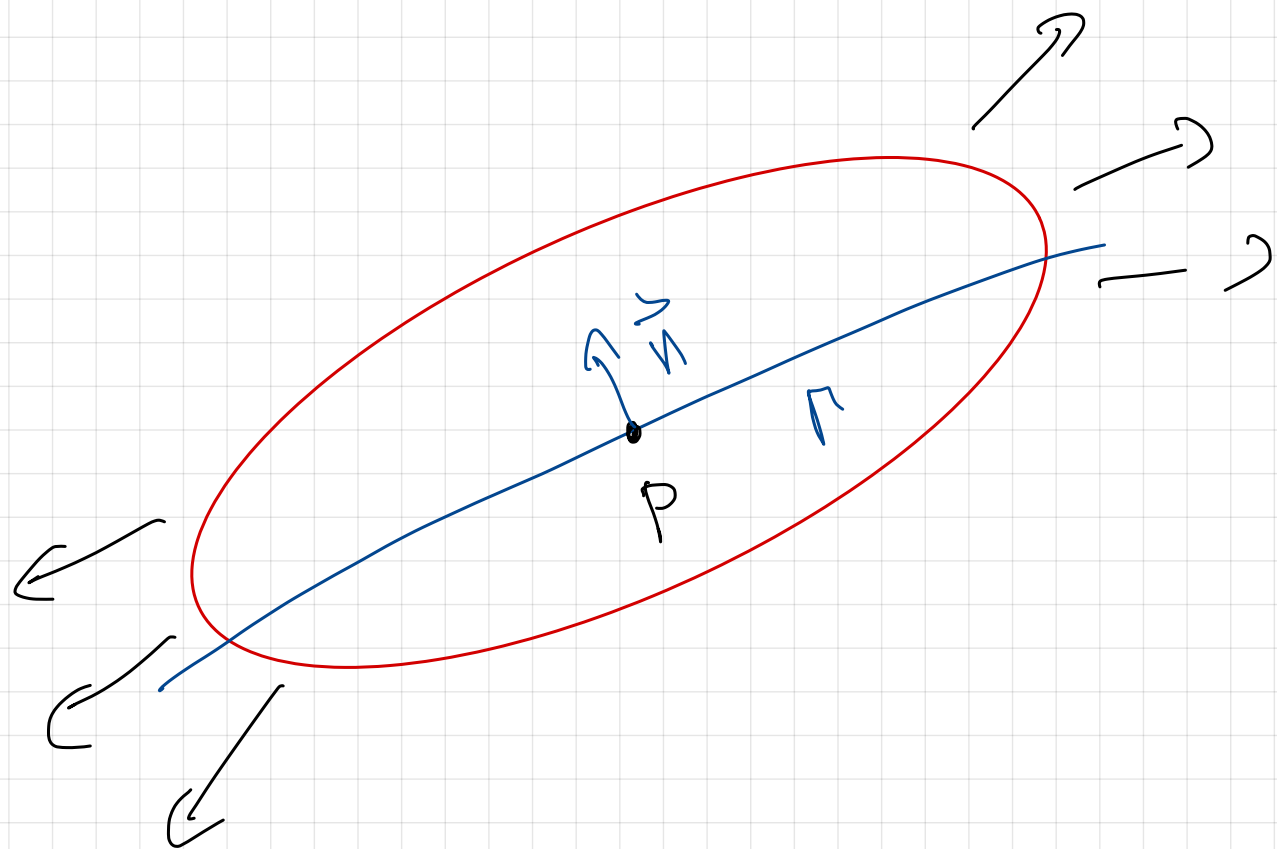
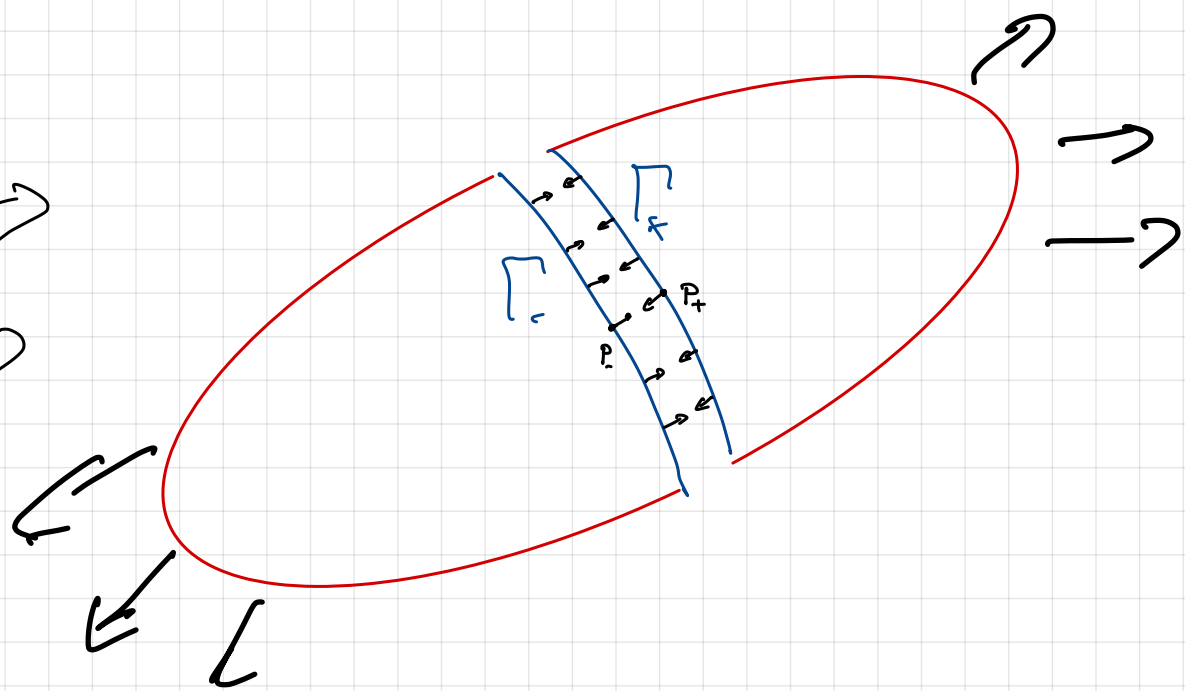
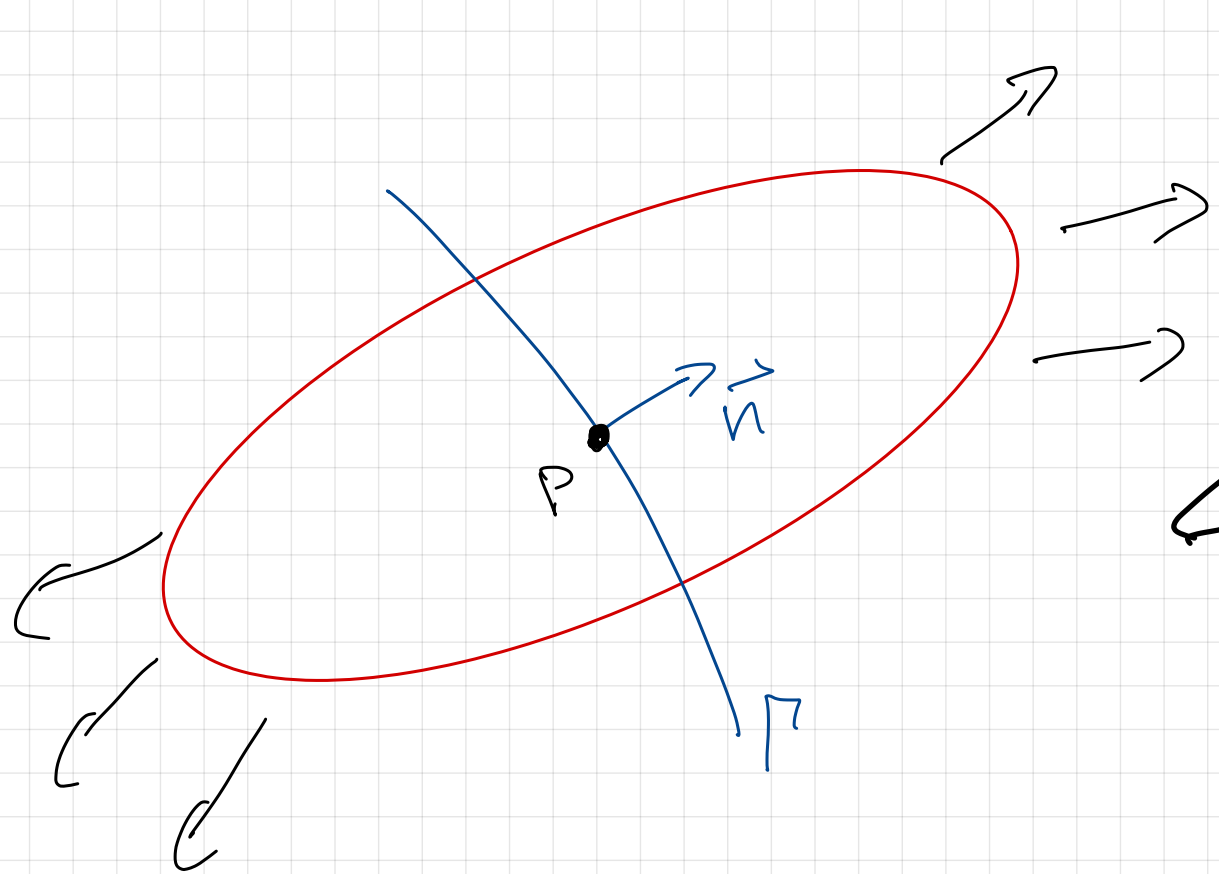
Cauchy's hypothesis: There exists a surface force density $\mathbf{t}_n = \mathbf{t}(\mathbf{x}, \mathbf{n}(\mathbf{x}))$ called the *traction*, defined for each vector \mathbf{n} such that on an oriented surface $\Gamma \subset \mathcal{B}$ with positive normal \mathbf{n} at \mathbf{x} , $\mathbf{t}(\mathbf{x}, \mathbf{n}(\mathbf{x}))$ is the force per unit area exerted across Γ upon the material on the negative side of Γ by the material on the positive side of Γ .







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traction \vec{T}
 depends only on
 position \vec{x}
 and direction \vec{n}

1.1.2 Momenta and Euler's laws

Consider $\Omega \subseteq \mathcal{B}$.

Total forces and torques:

$$\mathbf{F}(\Omega) = \mathbf{F}_{\text{body}}(\Omega) + \mathbf{F}_{\text{contact}}(\Omega) = \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{b}(\mathbf{x}, t) \, dv + \int_{\partial\Omega} \mathbf{t}_n \, da \quad (5)$$

$$\mathbf{G}(\Omega, \mathbf{0}) = \mathbf{G}_{\text{body}}(\Omega, \mathbf{0}) + \mathbf{G}_{\text{contact}}(\Omega, \mathbf{0}) = \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{x} \times \mathbf{b}(\mathbf{x}, t) \, dv + \int_{\partial\Omega} \mathbf{x} \times \mathbf{t}_n \, da. \quad (6)$$

Define $\mathbf{M}(\Omega)$: *total linear momentum* of Ω , and $\mathbf{H}(\Omega, \mathbf{0})$: *total angular momentum* of Ω with respect to the origin $\mathbf{0}$

$$\mathbf{M}(\Omega) = \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{v}(\mathbf{x}, t) \, dv \quad (7)$$

$$\mathbf{H}(\Omega, \mathbf{0}) = \int_{\Omega} \mathbf{x} \times (\rho(\mathbf{x}, t) \mathbf{v}(\mathbf{x}, t)) \, dv \quad (8)$$

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$$\mathbf{G}(\Omega, \mathbf{0}) = \mathbf{G}_{\text{body}}(\Omega, \mathbf{0}) + \mathbf{G}_{\text{contact}}(\Omega, \mathbf{0}) = \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{x} \times \mathbf{b}(\mathbf{x}, t) \, dv + \int_{\partial\Omega} \mathbf{x} \times \mathbf{t}_n \, da. \quad (10)$$

Define $\mathbf{M}(\Omega)$: *total linear momentum* of Ω , and $\mathbf{H}(\Omega, \mathbf{0})$: *total angular momentum* of Ω with respect to the origin $\mathbf{0}$

$$\mathbf{M}(\Omega) = \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{v}(\mathbf{x}, t) \, dv \quad (11)$$

$$\mathbf{H}(\Omega, \mathbf{0}) = \int_{\Omega} \mathbf{x} \times (\rho(\mathbf{x}, t) \mathbf{v}(\mathbf{x}, t)) \, dv \quad (12)$$

Euler's laws of motion: On any domain $\Omega \subseteq \mathcal{B}$

$$\frac{d\mathbf{M}}{dt} = \mathbf{F} \quad \frac{d\mathbf{H}}{dt} = \mathbf{G} \quad (13)$$

Euler's laws of motion: On any domain $\Omega \subseteq \mathcal{B}$

$$\frac{d\mathbf{M}}{dt} = \mathbf{F}, \quad \frac{d\mathbf{H}}{dt} = \mathbf{G} \quad (14)$$

Explicitly:

$$\underbrace{\frac{d}{dt} \int_{\Omega} \rho(\mathbf{x}, t) \mathbf{v}(\mathbf{x}, t) dv}_{\text{rate of change of linear momentum}} = \underbrace{\int_{\Omega} \rho(\mathbf{x}, t) \mathbf{b}(\mathbf{x}, t) dv + \int_{\partial\Omega} \mathbf{t}_n da}_{\text{sum of body and contact forces}}.$$

$$\underbrace{\frac{d}{dt} \int_{\Omega} \rho \mathbf{x} \times \mathbf{v} dv}_{\text{rate of change of angular momentum}} = \underbrace{\int_{\Omega} \rho \mathbf{x} \times \mathbf{b} dv + \int_{\partial\Omega} \mathbf{x} \times \mathbf{t}_n da}_{\text{torques due to body and traction forces}}.$$