

① (a) We want to compute  $\langle R^2 \rangle = \langle \left( \sum_{i=1}^N b t_i \right)^2 \rangle$

$$= \int_{(\mathbb{R}^3)^N} dV \left( \sum b t_i \right)^2 P(t_1, t_2, \dots, t_N) \quad \text{where } P \text{ is the prob.}$$

of the configuration [1]

Since each  $t_i$  is uniformly distributed over the unit sphere and indep. of all  $t_j, j \neq i$ , thus

$$= \int d\Omega_1 \dots \int d\Omega_N \left( \sum b t_i \right)^2 \cdot \frac{1}{(4\pi)^N} \quad \text{where } d\Omega_i = \sin \theta_i d\theta_i d\phi_i$$

[1] is solid angle.

Now we note

$$R^2 = (\sum r_i) \cdot (\sum r_j) = \sum_{i=1}^N r_i^2 + \sum_{i \neq j} r_i \cdot r_j \quad [2]$$

$$\langle r_i \cdot r_j \rangle \propto \underbrace{\int d\Omega_i t_i \cdot \int d\Omega_j t_j}_{\parallel} \quad \text{if } i \neq j$$

$$\int_0^{2\pi} \int_0^\pi t_i \sin \theta_i d\theta_i d\phi_i = 0 \quad \text{as } t_i = \begin{pmatrix} \sin \theta_i \cos \phi_i \\ \sin \theta_i \sin \phi_i \\ \cos \theta_i \end{pmatrix}$$

[2]

$$\text{And } \langle r_i^2 \rangle = \langle b^2 \rangle = b^2 \quad \text{since } \langle 1 \rangle = 1$$

by definition

[B]

$$\therefore \langle R^2 \rangle = \langle \sum r_i^2 \rangle = N b^2 \quad [1]$$

(b) We have internal energy  $E = \Gamma \sum \frac{b_i}{b} - 1$ .

The probability of a given chain  $\{\xi_1, \dots, \xi_N\}$

$$\therefore P(\{\xi_1, \dots, \xi_N\}) = \frac{1}{Z} \exp\left(-\frac{E(\{\xi_1, \dots, \xi_N\})}{k_b T}\right)$$

Here, since  $\xi_i = b_i \pm i$  and  $b_i$  can take any value from  $b$  to  $\infty$ , we have

$$\{2\} Z = \int_{(B_b)^N} e^{-\frac{E}{k_b T}} dV \quad \text{where } B_b = \{v : |v| \geq b\}$$

$$= \int d\xi_1 \int_{b_1}^{\infty} b_1^2 db_1 \cdots \int d\xi_N \int_{b_N}^{\infty} b_N^2 db_N \exp\left(\frac{\Gamma}{k_b T} \sum 1 - \frac{b_i}{b}\right)$$

$$= \prod_{i=1}^N \underbrace{\left( \int_0^{2\pi} \int_0^\pi \int_b^\infty b_i^2 \sin \theta_i e^{\frac{\Gamma}{k_b T} (1 - \frac{b_i}{b})} db_i d\theta_i d\varphi_i \right)}_{[2]}$$

$$4\pi e^{\frac{\Gamma}{k_b T}} \int_b^\infty b_i^2 e^{-\frac{b_i}{b}} db_i$$

$$- 4\pi e^{\frac{\Gamma}{k_b T}} \cdot 5b^3 e^{-1} \quad \text{using given formula}$$

$$\therefore Z = (20\pi e^{\frac{\Gamma}{k_b T} - 1} \cdot b^3)^N \quad [1]$$

(c) Now the energy consists of 2 parts :

$$(i) E = -F_z \cdot B + E_{\text{stretch}} = -F_z \sum b_i \cos \theta_i + \Gamma \sum \frac{b_i}{b} - 1$$

{2}

$$\text{Now } Z = \int_{(B_b)^N} dV \exp\left(-\frac{E}{k_b T}\right) = \int_{(B_b)^N} dV e^{\lambda_1 \sum \frac{b_i}{b} \cos \theta_i + \lambda_2 \sum 1 - \frac{b_i}{b}}$$

$$\lambda_1 = \frac{F_z b}{k_b T}, \lambda_2 = \frac{\Gamma}{k_b T}$$

$$\rightarrow Z = \prod_{i=1}^N \left( \int_0^{2\pi} \int_0^\pi \int_b^\infty b_i^2 \sin \theta_i e^{\lambda_2 - \frac{b_i}{b}(\lambda_2 - \lambda_1 \cos \theta_i)} db_i d\theta_i d\varphi_i \right)$$

(S/N)

$$2\pi e^{\lambda_2} \int_b^\infty \frac{b}{\lambda_1 b_i} b_i^2 e^{-\frac{b_i}{b}(\lambda_2 - \lambda_1 \cos \theta_i)} \Big|_{\theta=0}^\pi db_i$$

$$= \frac{2\pi b e^{\lambda_2}}{\lambda_1} \int_b^\infty b_i e^{-\frac{b_i}{b}(\lambda_2 + \lambda_1)} - b_i e^{-\frac{b_i}{b}(\lambda_2 - \lambda_1)} db_i$$

↑

this will diverge if  $\lambda_1 > \lambda_2$

∴ if  $F_z b > \Gamma$

{2}

ii) If  $\lambda_1 < \lambda_2$ , we obtain (using provided integ. formulas with  
 $a = \lambda_2 - \lambda_1, c = b$ )

$$Z = \left( \frac{2\pi b e^{\lambda_2}}{\lambda_1} \left| \frac{(1+\lambda_2+\lambda_1)b^2}{(\lambda_2+\lambda_1)^2 e^{\lambda_2-\lambda_1}} - \frac{(1+\lambda_2-\lambda_1)b^2}{(\lambda_2-\lambda_1)^2 e^{\lambda_2+\lambda_1}} \right| \right)^N$$

$$\stackrel{\text{call}}{=} f(\lambda_1, \lambda_2)^N$$

[2]

Now,  $R_z = \sum_{i=1}^N b_i \cos \theta_i$ , so

$$\langle R_z \rangle = \frac{1}{Z} \int \left( \sum b_i \cos \theta_i \right) e^{\lambda_1 \sum \frac{b_i}{b} \cos \theta_i + \lambda_2 \sum 1 - \frac{b_i}{b}} dV$$

(B<sub>b</sub>)<sup>N</sup> [1]

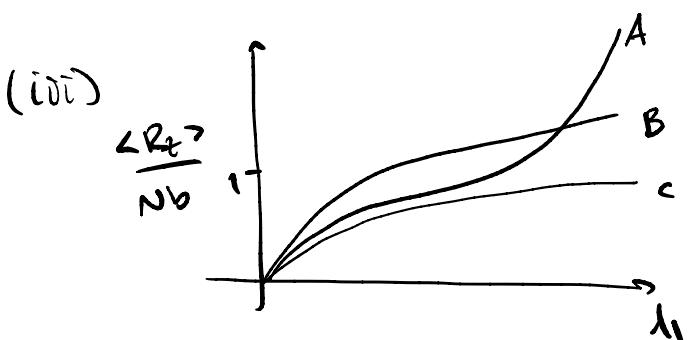
Observe  $\frac{\partial}{\partial \lambda_1} Z = \int \frac{1}{b} \sum b_i \cos \theta_i e^{\lambda_1 \dots} dV$

(B<sub>b</sub>)<sup>N</sup>

$$\therefore \langle R_z \rangle = b \cdot \frac{1}{Z} \frac{\partial}{\partial \lambda_1} Z = b \frac{\partial}{\partial \lambda_1} \log Z$$

[2]

{S}  $= b N \frac{\partial}{\partial \lambda_1} \log f$  with  
 $f$  defined as above



[1] A  $\Rightarrow$  ii) - linear energy

B  $\Rightarrow$  iii) - quadratic stretching energy

C  $\Rightarrow$  i) inextensible

- the inextensible chain asymptotes to 1 as

[1] pulling force  $\lambda_1 \rightarrow \infty$  -  $\langle R_z \rangle$  only reaches  $N_b$  if chain  $\Rightarrow$  straight

- the linear energy leads to a divergence

in  $\langle R_z \rangle$ , as we showed, as  $\lambda_1 \rightarrow \lambda_2 = 10$

[1]

(linear energy  $\Rightarrow$  a constant resistance force,

} so nothing stops the extension if  $bF_z > 0$   
- this  $\Rightarrow$  not physically realistic)

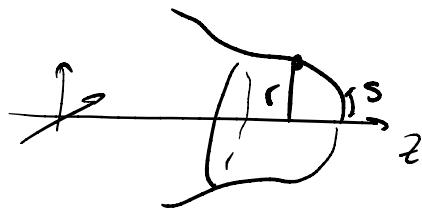
More than  
expected  
from  
students

- For quadratic energy,  $\langle R_z \rangle$  doesn't diverge  
→ (linear stretching force  $\Rightarrow$  resistance increases w/ extension), but  $\langle R_z \rangle$  can exceed  $N_b$  with sufficient  $\lambda_1$

[N]

(2)

a)



we parameterize the surface as

$$\underline{x}(s, \varphi) = \begin{pmatrix} r(s) \cos \varphi \\ r(s) \sin \varphi \\ z(s) \end{pmatrix}$$

$$\rightarrow \underline{x}_s = \begin{pmatrix} r' \cos \varphi \\ r' \sin \varphi \\ z' \end{pmatrix}, \quad \underline{x}_\varphi = \begin{pmatrix} -r \sin \varphi \\ r \cos \varphi \\ 0 \end{pmatrix} \quad [2]$$

$$\Rightarrow \text{Metric tensor has } g_{11} = \underline{x}_s \cdot \underline{x}_s = r'^2 + z'^2$$

[1]

$$g_{12} = 0, \quad g_{22} = \underline{x}_\varphi \cdot \underline{x}_\varphi = r^2$$

$$\text{Normal vec } \underline{n} = \frac{\underline{x}_s \wedge \underline{x}_\varphi}{\|\underline{x}_s \wedge \underline{x}_\varphi\|}. \quad \underline{x}_s \wedge \underline{x}_\varphi = \begin{pmatrix} -rz' \cos \varphi \\ -rz' \sin \varphi \\ rr' \end{pmatrix}$$

$$\Rightarrow \underline{n} = \frac{(-rz' \cos \varphi, -rz' \sin \varphi, rr')^T}{r(r'^2 + z'^2)^{1/2}}. \quad [1]$$

$$\text{Curvature tensor: } K_{11} = -\underline{n} \cdot \frac{\partial^2 \underline{x}}{\partial s^2} = \begin{pmatrix} r'' \cos \varphi \\ r'' \sin \varphi \\ z'' \end{pmatrix}$$

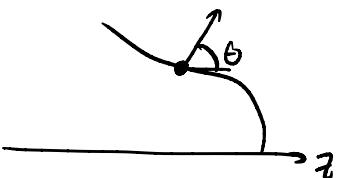
$$= \frac{-1}{r(r'^2 + z'^2)^{1/2}} \cdot (r'r''z'' - r''r'z')$$

[2]

$$K_{22} = -\underline{n} \cdot \frac{\partial^2 \underline{x}}{\partial \varphi^2} = \frac{-r^2 z'}{r(r'^2 + z'^2)^{1/2}} \quad (\because K_{12} = K_{21} = 0)$$

$$\begin{pmatrix} -r \cos \varphi \\ -r \sin \varphi \\ 0 \end{pmatrix}$$

by axial symmetry  
 $\varphi$  and  $s$  are principal directions) [1]



we note  $r' = \cos\theta$ ,  $z' = -\sin\theta$

so  $r'^2 + z'^2 = 1 \sim$  previous

formulas.

$\lambda_s, \lambda_q$  are the eigenvals of  $L = G^{-1}K$

$$G^{-1} = \frac{1}{g_{11} g_{22}} \begin{pmatrix} g_{22} & 0 \\ 0 & g_{11} \end{pmatrix}, \quad K = \begin{pmatrix} K_{11} & 0 \\ 0 & K_{22} \end{pmatrix} \quad [1]$$

$$\Rightarrow L \text{ is diag w/ } L_{11} = \frac{K_{11}}{g_{11}} = \bar{z}'r'' - r'\bar{z}''$$

[2]

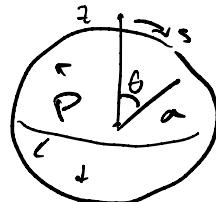
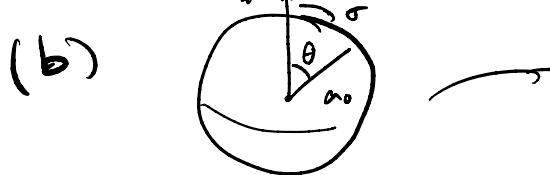
$$\& L_{22} = \frac{K_{22}}{g_{22}} = -\frac{r\bar{z}'}{r^2}$$

$$\text{Now } r'' = -\sin\theta \theta', \quad \bar{z}'' = -\cos\theta \theta'$$

$$\Rightarrow L_{11} = \lambda_s = \theta'(s), \quad L_{22} = \lambda_q = \frac{\sin\theta}{r}, \quad [1]$$

as desired.

(B/S)



We have in initial state:

$$\left. \begin{array}{l} \sigma = a_0 \theta \\ r_0 = a_0 \sin \theta \\ z_0 = a_0 \cos \theta \end{array} \right\} \xrightarrow{\text{deforms}} \left. \begin{array}{l} s = a \theta \\ r = a \sin \theta \\ z = a \cos \theta \end{array} \right\}$$

[2]

The principal stretches are defined by  $\lambda_s = \frac{ds}{d\sigma}$ ,  $\lambda_r = \frac{r}{r_0}$

→ here we have  $s = \frac{a}{a_0} \theta \Rightarrow \lambda_s = \frac{a}{a_0}$  &  $\lambda_r = \frac{a \sin \theta}{a_0 \sin \theta} = \frac{a}{a_0}$ .

We're given  $t_s = t_r = f\left(\frac{a}{a_0}\right)$

- $f(1) = 0$  so that no stress in ref. state

[2] 

- $f'(1) > 0$  so that the material is in tension ( $t_s = t_r > 0$ ) when stretched ( $a > a_0$ )

$\frac{\partial t_s}{\partial s} = \frac{\cos \theta}{r} (t_s - t_r)$  automatically satisfied, so turn to

$P = t_s \kappa_s + t_r \kappa_r$  . We have  $\kappa_s = \frac{ds}{ds} = \frac{1}{a}$ ,  $\kappa_r = \frac{dr}{ds} = \frac{1}{a}$

(obvious for a sphere, but should verify the relatives hold)

[2] Thus,  $P = \frac{2}{a} f\left(\frac{a}{a_0}\right)$  defines the pressure-radius relation.

[S]

For a fluid biomembrane, we have the

$$\text{energy } E = \int_{\Sigma} dS (\gamma + 2\pi H^2) - VP$$

Lagrange multiplier to  
enforce pressure constraint.

For sphere radius  $a$ ,  $H = \frac{1}{a}$ ,  $dS = 4\pi a^2$ ,

$$\text{so } E(a) = 4\pi \gamma a^2 + 4\pi \frac{a^2}{a^2} \kappa - \frac{4}{3}\pi a^3 P$$

[2]

we find  $a$  via  $E'(a) = 0 \rightarrow 8\pi \gamma a - 4\pi a^2 P = 0$

{S}  $\Rightarrow \text{pressure-radius relation } | P = \frac{2\gamma}{a} |$

- this agrees with (ii) if  $f(\frac{a}{a_0}) = \gamma$

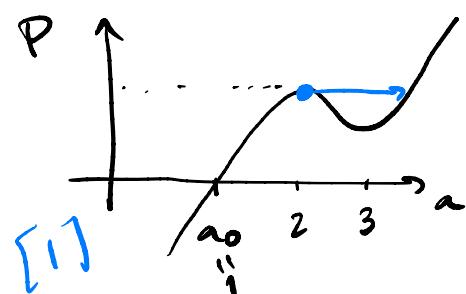
constant

[1]

(iv) Inflation instability: Noting that  $P(k) = 2 \frac{f(\frac{a}{a_0})}{a}$

(w/  $a_0=1$ ), we are given that

$$\frac{f'(a)}{a} - \frac{f(a)}{a^2} = \frac{d}{da} \left( \frac{f(a)}{a} \right) = \frac{1}{a} \left( \frac{P(a)}{2} \right)$$



has roots at  $a=2, a=3$

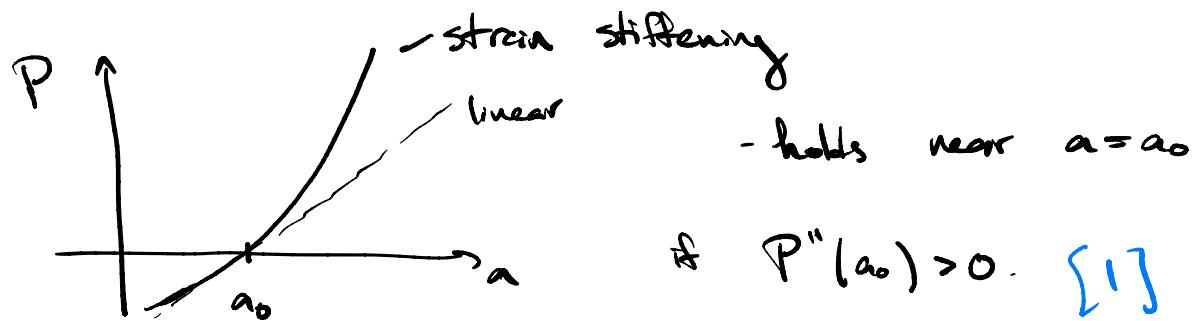
$\Rightarrow P(a)$  has shape at left

A limit point instability will occur as

$$P_B \text{ increased past } P(2) = \frac{2f(2)}{2}$$

[1]

(v) Strain-stiffening response will mean as  $a$  is increased, greater increase in  $P$  is needed to further inflate.



$$\text{we have } P'(a) = -\frac{2}{a^2} f\left(\frac{a}{a_0}\right) + \frac{2}{aa_0} f'\left(\frac{a}{a_0}\right)$$

$$\rightarrow P''(a) = -\frac{4}{a^3} f\left(\frac{a}{a_0}\right) - \frac{4}{aa_0^2} f'\left(\frac{a}{a_0}\right) + \frac{2}{aa_0^2} f''\left(\frac{a}{a_0}\right)$$

$$\Rightarrow P''(a_0) = -\frac{4}{a_0^3} f'(1) + \frac{2}{a_0^3} f''(1) \quad (\text{as } f(1) = 0)$$

so strain stiffening observed if  $\underline{\underline{|f''(1)| > 2|f'(1)|}}$

[N]

[2]

$$(3) \quad \underline{r}' = \alpha \gamma \underline{d}_3, \quad \underline{d}'_i = \gamma \underline{u} \wedge \underline{d}_i, \quad \underline{n} + \underline{f} = \underline{0}, \quad \underline{m} + \underline{r} \wedge \underline{n} = \underline{0}$$

$$\text{w/ } \underline{m} = k_1 \underline{u}_1 \underline{d}_1 + k_2 \underline{u}_2 \underline{d}_2 + k_3 \underline{u}_3 \underline{d}_3, \quad \underline{n} \cdot \underline{d}_3 = k_4 (\alpha - 1)$$

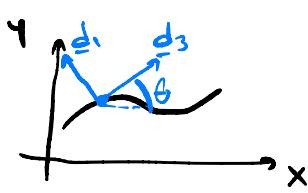
(a)  $\underline{r}$  - rod centreline,  $\underline{u}$  curvature vector

$\{\underline{d}_1, \underline{d}_2, \underline{d}_3\}$  material frame, gives rods orientation

$\underline{n}$  - resultant force  $\sim$  rd

[1]

$\underline{m}$  - resultant moment



Let  $\underline{r} = \begin{pmatrix} x \\ y \\ 0 \end{pmatrix}$ , and  $\theta$  the angle between tangent  $\underline{d}_3$  and x-axis

$$\text{so } \underline{d}_3 = \begin{pmatrix} \cos \theta \\ \sin \theta \\ 0 \end{pmatrix}. \quad \text{Choose } \underline{d}_2 = \underline{e}_2, \text{ so } \underline{d}_1 = \begin{pmatrix} -\sin \theta \\ \cos \theta \\ 0 \end{pmatrix} \quad [2]$$

$$\text{Then } \underline{d}'_3 = \theta'(S_0) \underline{d}_1 = \gamma \underline{u} \wedge \underline{d}_3 = \gamma (u_2 \underline{d}_1 - u_1 \underline{d}_2)$$

$$\Rightarrow u_1 = 0, \quad u_2 = \frac{1}{\gamma} \theta'. \quad \text{Also, } \underline{d}'_2 = 0 = \gamma (u_1 \underline{d}_3 - u_3 \underline{d}_1)$$

$$\Rightarrow u_3 = 0. \quad \text{So } \underline{m} = m_2 \underline{d}_2 = K_2 \frac{1}{\gamma} \theta' \underline{e}_2 \quad [3]$$

$$\text{Let } \underline{u} = \begin{pmatrix} u_x \\ u_y \\ 0 \end{pmatrix}, \quad \underline{f} = \begin{pmatrix} f \\ g \\ 0 \end{pmatrix} \Rightarrow \left| \begin{array}{l} u'_x + f = 0 \\ u'_y + g = 0 \end{array} \right| \quad \begin{array}{l} (1) \\ (2) \end{array}$$

$$\underline{r}' = \alpha \gamma \underline{d}_3 \rightarrow \left| \begin{array}{l} x' = \alpha \gamma \cos \theta \\ y' = \alpha \gamma \sin \theta \end{array} \right| \quad \begin{array}{l} (3) \\ (4) \end{array} \quad \text{Moment balance}$$

$$m'_2 \underline{e}_2 + \alpha \gamma \underline{d}_3 \wedge \underline{u} = 0$$

$$\Rightarrow \left| K_2 \left( \frac{1}{\gamma} \theta' \right)' + \alpha \gamma (u_y \cos \theta - u_x \sin \theta) = 0 \right| \quad [5]$$

[2]

$$\& \quad \underline{n} \cdot \underline{d}_3 = \left| u_x \cos \theta + u_y \sin \theta = k_4 (\alpha - 1) \right| \quad [6]$$

(b) We take  $f = 0$ ,  $g = -\frac{\kappa}{8}\gamma$ .

For flat state,  $\gamma = 0 \equiv \theta$ , so (b)  $\Rightarrow n_x = k_y(\alpha - 1)$ .

(3), (4)  $\rightarrow x' = \alpha\gamma$ ,  $\gamma' = 0$ . (1), (2)  $\rightarrow n_x, n_y$  constant

And (5)  $\rightarrow n_y = 0$ .

[2]

Bdy cond are : At  $S_0 = 0$ ,  $x = 0$ ,  $\gamma = 0$ ,  $\underline{m} = 2 \Rightarrow g' = 0$

[S] At  $S_0 = L_0$ ,  $x = L_0$ ,  $\gamma = 0$ ,  $\theta' = 0$

$\therefore x = S_0 \Rightarrow x' = \alpha\gamma = 1 \Rightarrow$  must have  $\alpha = \frac{1}{8}$  [2]

Thus, given flat state given by  $\begin{cases} x^{(0)} = S_0 \\ n_x^{(0)} = k_y \left( \frac{1}{8} - 1 \right) \\ \text{all others} = 0 \end{cases}$

(c) Perturb :  $x \sim x^{(0)} + \epsilon x^{(1)}$ ,  $\gamma \sim \epsilon \gamma^{(1)}$ ,  $\theta \sim \epsilon \theta^{(1)}$ ,  $n_y \sim \epsilon n_y^{(1)}$ ,

$n_x \sim n_x^{(0)} + \epsilon n_x^{(1)}$ ,  $\alpha \sim \frac{1}{8} + \epsilon \alpha^{(1)}$ .

Then (2)  $\rightarrow n_y^{(0)} = \frac{\kappa}{8} \gamma^{(0)}$ , (4)  $\rightarrow \gamma^{(0)} = \theta^{(0)}$

(5)  $\rightarrow K_2 \theta^{(0)} + \gamma \left( n_y^{(0)} - n_x^{(0)} \theta^{(0)} \right) + O(\epsilon) = 0$

$\stackrel{\frac{\partial}{\partial \theta}}{\Rightarrow} K_2 \gamma^{(0)(0)} + \gamma \left( \frac{\kappa}{8} \gamma^{(0)} - k_y \left( \frac{1-\gamma}{8} \right) \gamma^{(0)} \right) = 0$  [2]

$\Rightarrow K_2 \gamma^{(0)(0)} + K_y (\gamma - 1) \gamma^{(0)(0)} + K \gamma^{(0)(0)} = 0$

BC  $\gamma^{(0)(0)} = \gamma^{(0)(0)} = 0$  at  $S_0 = 0, L_0$ .

$\theta^{(0)(0)}$

• we seek smallest  $\gamma = \gamma^* > 1$  at which  $\gamma^{(1)}$  has a non-trivial soln. Taking  $\gamma^{(1)} = ce^{i\omega_0 t}$

$$\rightarrow K_2 \omega^4 - K_4(\gamma-1)\omega^2 + \kappa = 0$$

$$\Rightarrow \omega^2 = \frac{K_4(\gamma-1) \pm \sqrt{K_4^2(\gamma-1)^2 - 4K_2\kappa}}{2K_2} =: \omega_{\pm}^2$$

[1]

• for oscillatory soln, must have  $\omega_{\pm}^2 > 0$

$$\rightarrow (\gamma^*-1)^2 > \frac{4K_2\kappa}{K_4}. \text{ In this range, we'll have}$$

$$\gamma^{(1)} = A \cos \omega_+ \delta_0 + B \sin \omega_+ \delta_0 + C \cos \omega_- \delta_0 + D \sin \omega_- \delta_0$$

$$\gamma^{(1)}(0) = A + B = 0 \quad \left. \right\} \Rightarrow A = B = 0 \quad [1]$$

$$\gamma^{(1)''}(0) = -A\omega_+^2 - B\omega_-^2 = 0$$

$$\text{Then } \gamma^{(1)}(L_0) = C \sin \omega_+ L_0 + D \sin \omega_- L_0 = 0$$

$$\gamma^{(1)''}(L_0) = -C\omega_+^2 \sin \omega_+ L_0 - D\omega_-^2 \sin \omega_- L_0 = 0$$

Non-trivial solns if  $\sin \omega_+ L_0 = n\pi, n \in \mathbb{N}$  [2]

$$\text{or } \sin \omega_- L_0 = n\pi$$

Critical buckling is defined by smallest

$$\gamma^* > 1 + \frac{2\sqrt{K_2\kappa}}{\kappa} \text{ satisfying}$$

[2]

one of the above.

(d) On an infinite domain (no BC),

$$(\gamma^* - 1)^2 = 4 \frac{K_2 \kappa}{K_1} \quad (\text{smallest } \gamma \text{ for which oscillatory solutions exist})$$

$$\rightarrow \gamma^* - 1 \sim \sqrt{\kappa}$$

[1]

$$\text{And } \omega^* = \left( \frac{K_1 (\gamma^* - 1)}{2K_2} \right)^{\frac{1}{2}} = \frac{\kappa^{\frac{1}{4}}}{K_2^{\frac{1}{4}}} \quad [2]$$

$$\text{Wavelength } \lambda = \frac{2\pi}{\omega} \Rightarrow \lambda^* \sim \kappa^{-\frac{1}{4}}. \quad [N]$$

As  $\kappa$  increases, foundation provides increased resistance, so more growth needed to trigger instability, and observed wavelength decreases, because it costs more to have large deformation (foundation energy) relative to bending and stretching. [2]